

2.004

Control Theory

error = Desired - actual

$$\frac{d^2y}{dt^2} + 2\zeta\omega_n \frac{dy}{dt} + \omega_n^2 y = 0$$

$e = 2.71828$

$$AV' + BV = C(t)$$

$e^{-1} = 0.3679$

$$\zeta = \frac{A}{B}$$

$e^{-2} = 0.1353$

$e^{-3} = 0.0498$

$e^{-4} = 0.0183$

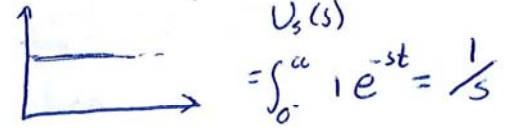
$$\mathcal{L}\{f(t)\} = \int_0^\infty f(t) e^{-st} dt$$

$$s = \sigma + j\omega$$

$j = \sqrt{-1}$

$$\mathcal{L}\{f'(t)\} = sF(s) - f(0^-)$$

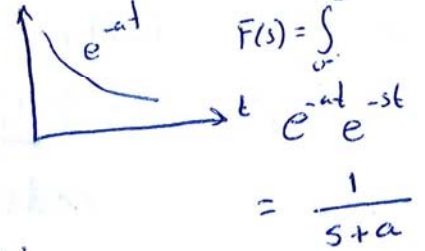
Unit Step



$$\mathcal{L}\{f''(t)\} = s^2 F(s) - sf'(0^-) - f(0^-)$$

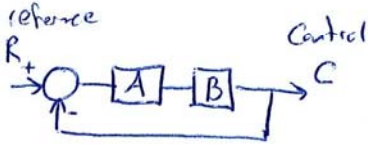
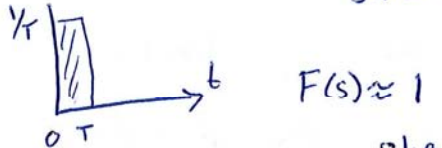
$$\mathcal{L}\left\{\frac{d^m f}{dt^m}\right\} = s^m F(s) - \sum_{k=1}^m s^{m-k} f^{(k-1)}(0^-)$$

one sided exp



$$\mathcal{L}\left\{\int_0^t f(\tau) d\tau\right\} = \frac{1}{s} F(s)$$

$f(t) = e^{-at}$



$\frac{C}{R} = TF = \frac{AB}{1+AB}$

State Space

$$\dot{X} = AX + BU$$

$$Y = CX + DU$$

to transform Function

Final Value Thm

$$\lim_{t \rightarrow \infty} f(t) = \lim_{s \rightarrow 0} s F(s)$$

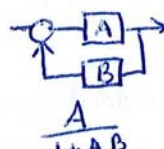
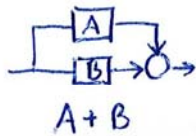
use to compute steady state

Short Impulse

$$sX(s) = AX(s) + BU(s)$$

$$X(s) = (sI - A)^{-1} BU(s)$$

$$Y(s) = C(sI - A)^{-1} B + D$$



Matrix Manipulation

$$A^{-1} = \frac{\text{adj}[A]}{\det[A]}$$

Flip mult -1

$$A = \begin{bmatrix} a_{11} & a_{12} \\ a_{21} & a_{22} \end{bmatrix} \rightarrow \text{adj}A = \begin{bmatrix} a_{22} & -a_{12} \\ -a_{21} & a_{11} \end{bmatrix}$$

$$V(s) = \frac{1}{a_n s^n + a_{n-1} s^{n-1} + \dots + a_1 s + a_0} U(s)$$

$$N(s) = b_m s^m + b_{m-1} s^{m-1} + \dots + b_0$$

$$Y = CX + DU$$

$$= [b_0 \ b_1 \ b_2 \ \dots] \begin{bmatrix} X_1 \\ X_2 \\ X_3 \\ \vdots \\ X_n \end{bmatrix} + [D] U$$

Laplace Transforms

$\delta(t) \rightarrow 1$

$u(t) \rightarrow 1/s$

$t u(t) \rightarrow 1/s^2$

$t^n u(t) \rightarrow \frac{n!}{s^{n+1}}$

$e^{-at} u(t) \rightarrow \frac{1}{s+a}$

$\sin \omega t u(t) \rightarrow \frac{\omega}{s^2 + \omega^2}$

$\cos \omega t u(t) \rightarrow \frac{s}{s^2 + \omega^2}$

$$\begin{bmatrix} \dot{X}_1 \\ \dot{X}_2 \\ \vdots \\ \dot{X}_n \end{bmatrix} = \begin{bmatrix} 0 & 1 & 0 & \dots & 0 \\ 0 & 0 & 1 & \dots & 0 \\ \vdots & \vdots & \vdots & \ddots & \vdots \\ 0 & 0 & 0 & \dots & -a_1 \\ -\frac{a_0}{a_n} & \frac{a_1}{a_n} & \dots & -\frac{a_{n-1}}{a_n} & 0 \end{bmatrix} \begin{bmatrix} X_1 \\ X_2 \\ \vdots \\ X_n \end{bmatrix} + \begin{bmatrix} 0 \\ 0 \\ \vdots \\ 1/a_n \end{bmatrix} U$$

"Phase Variable" form of state variables.

$$\frac{X(s)}{U(s)} = \frac{1}{s^3 + 6s^2 + 5s + 10K}$$

$$\frac{Y(s)}{U(s)} = 10Ks + 10B$$

$$\dot{X} = \begin{bmatrix} 0 & 1 & 0 \\ 0 & 0 & 1 \\ -10K & -5 & -6 \end{bmatrix} X + \begin{bmatrix} 0 \\ 0 \\ 1/10 \end{bmatrix} U(t)$$

low to High phase variable state space representation

$$Y = [10B \ 10K \ 0] \begin{bmatrix} X_1 \\ X_2 \\ X_3 \end{bmatrix} + [0] U(t)$$

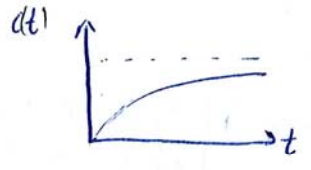
Velocity	mass	Dampers	Springs	Forces
	$\curvearrowright \dot{V}_m = \frac{1}{m} F_m$ node	$\curvearrowright V_B = \frac{1}{B} F_B$ $\curvearrowright F_B = B V_B$	$\curvearrowright \dot{F}_K = K V_K$ loop add K \sum loop	

Voltage	Capacitors	Resistors	Inductors	Currents
	$\curvearrowright \dot{V}_C = \frac{1}{C} \dot{I}_C$ node	$\curvearrowright V_{RL} = R \dot{I}_{RL}$ $\curvearrowright \dot{I}_{RS} = \frac{1}{R_S} V_{RS}$	$\curvearrowright \dot{I}_L = \frac{1}{L} V_L$ loop	

Vel	Inertia	Friction	Springs	Torques
	$\curvearrowright \dot{\Omega}_J = \frac{1}{J} T_J$ node	$\curvearrowright \Omega_B = \frac{1}{B} T_B$ $\curvearrowright T_B = B \Omega_B$	$\curvearrowright \dot{T}_K = K \Omega_K$ loop	

Pressure	Inertia	Friction	Moving Mass	Flow Rate
	$\curvearrowright \dot{P}_{cf} = \frac{1}{C_f} Q_{cf}$ node	$\curvearrowright P = R_f Q$ $\curvearrowright Q = \frac{1}{R_f} P$	$\curvearrowright \dot{Q} = \frac{1}{I_f} P$ loop	

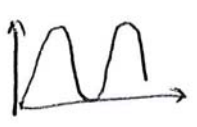
Overdamped responses
Eigen Value: $-\sigma_1, -\sigma_2$
 $c(t) = K_1 e^{-\sigma_1 t} + K_2 e^{-\sigma_2 t}$



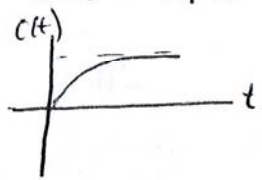
Underdamped responses
Eigen Values: $-\sigma_d \pm j\omega_d$
 $c(t) = A e^{-\sigma_d t} \cdot \cos(\omega_d t - \phi)$



Undamped Responses
Eigen Value $\pm j\omega_1$
 $c(t) = A \cos(\omega_1 t - \phi)$



Critically damped
Eigen Value $-\sigma_1, -\sigma_1$
 $c(t) = K_1 e^{-\sigma_1 t} + K_2 t e^{-\sigma_1 t}$



$$\omega_n^2 = \frac{1}{LC}$$

$$G(s) = \frac{b}{s^2 + as + b}$$

$$\omega_n = \sqrt{b}$$

$$\zeta = \frac{a/2}{\omega_n} = \frac{a}{2\sqrt{b}}$$

$$T_p = \frac{\pi}{\omega_n (1 - \zeta^2)^{1/2}}$$

time to reach peak

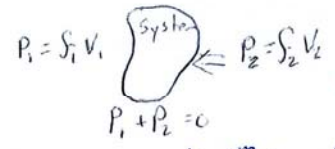
$$\zeta = \frac{-\ln(\%OS/100)}{(\pi^2 + \ln^2(\%OS/100))^{1/2}}$$

$$\%OS = e^{-(\zeta \pi / \sqrt{1 - \zeta^2})} \times 100$$

percent over shoot.

$$T_s = \frac{4}{\zeta \omega_n}$$

Settling time to reach $\pm 2\%$ steady state



Transformer
 $V_1 \propto V_2$
 $I_1 \propto I_2$
 one of each.

Gyator
 $V_1 \propto I_2$
 $I_1 \propto V_2$
 Both in or both out



- Normal tree
1. Access Variable sources
 2. A type elements
 3. Twoports elements
 4. D type elements
 5. T type elements
- b. No through var sources

TF. $G(s) = \frac{b_n s^n + \dots + b_1 s + b_0}{a_n s^n + \dots + a_1 s + a_0}$

$\rightarrow [b_n s^n + \dots + b_1 s + b_0] X(s) \rightarrow [a_n s^n + \dots + a_1 s + a_0] Y(s)$

$i_{Lm} = \frac{1}{L} \int V_{Lm}$
 $V_{Rm} = R_m i_{Rm}$
 $T_B = B \Omega_B$
 $V_1 = \frac{1}{K_a} \Omega_2$
 $T_2 = -\frac{1}{K_a} i_1$

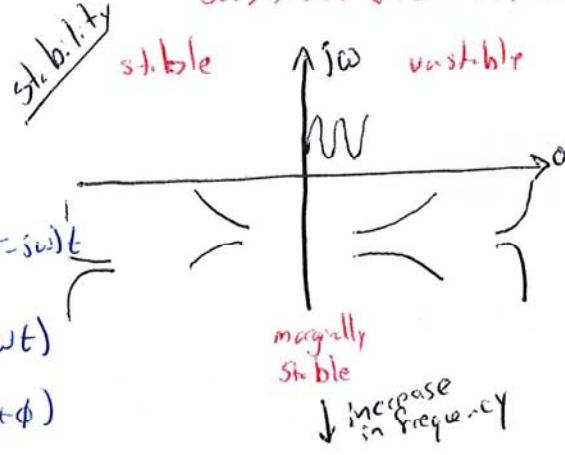
Method of undetermined Coef

$U(t)$	$Y_p(t)$
K	K
Kt^n	$K_n t^n + K_{n-1} t^{n-1} + \dots$
$K e^{at}$	$K e^{at}$
$K e^{j\omega t}$	$K e^{j\omega t}$
$K \cos(\omega t)$	$K_1 \cos \omega t + K_2 \sin \omega t$
$K \sin(\omega t)$	$K_1 \cos \omega t + K_2 \sin \omega t$

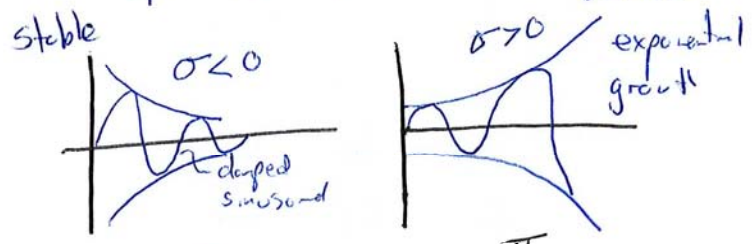
solve ode
 $a_n \frac{d^n y}{dt^n} + a_{n-1} \frac{d^{n-1} y}{dt^{n-1}} + \dots + a_1 \frac{dy}{dt} + a_0 y = X(t)$ ode
 Char. Eqn: $(a_n \lambda^n + a_{n-1} \lambda^{n-1} + \dots + a_1 \lambda + a_0) = 0$
 $Y_h(t) = \sum_{k=1}^n C_k e^{\lambda_k t}$

$G(s) = \frac{N(s)}{D(s)}$
 $D(s) \Rightarrow \det(sI - A) = 0$

Euler
 $\cos(\omega t) = \frac{1}{2} (e^{j\omega t} + e^{-j\omega t})$
 $\sin(\omega t) = \frac{1}{2j} (e^{j\omega t} - e^{-j\omega t})$



$\lambda = a \pm jb$
 $(a + j\omega) e^{(a + j\omega)t} + (a - j\omega) e^{(a - j\omega)t}$
 $z a e^{\sigma t} \cos(\omega t) + z b e^{\sigma t} \sin(\omega t)$
 $y(t) = z \sqrt{a^2 + b^2} e^{-\sigma t} \sin(\omega t + \phi)$
 $\phi = \tan^{-1}(a/b)$



$\zeta = \frac{\text{exponential decay frequency}}{\text{Natural frequency}}$

$G(s) = \frac{b}{s^2 + as + b}$; $G(s) = \frac{b}{s^2 + b}$

$T_p = \frac{\pi}{\omega_n \sqrt{1 - \zeta^2}} = \frac{\pi}{\omega_d}$

5% $\approx \frac{\sqrt{2}}{2} = \zeta$
 0.707

$\omega_n = \sqrt{b}$
 $\zeta = \frac{a/2}{\omega_n} = \frac{a}{2\sqrt{b}}$

$\%OS = e^{-(\zeta \pi / \sqrt{1 - \zeta^2})} \times 100$

$C_{fund} = \frac{K}{\omega_n^2}$

$G(s) = \frac{\omega_n^2}{s^2 + 2\zeta \omega_n s + \omega_n^2}$

$\zeta = \frac{-\ln(\%OS/100)}{\sqrt{\pi^2 + \ln^2(\%OS/100)}}$

$G(s) = \frac{b_n s^n + \dots + b_1 s + b_0}{a_n s^n + \dots + a_1 s + a_0}$

$= k + \frac{N'(s)}{D(s)}$ $k = \frac{b_n}{a_n}$

$G(s) = \frac{s+b}{s+a} = 1 - \frac{b-a}{s+a}$

$T_s = \frac{4}{\zeta \omega_n}$

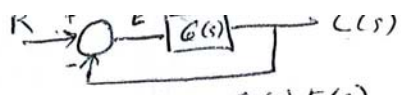
$G(s) = \frac{a/b}{(s+a)(s+b)}$

$y_{step} = U_s(t) + \frac{b-a}{a} (1 - e^{-at})$

$y_{step} = 1 - \frac{b(a-c)}{c(a-b)} e^{-at} + \frac{a(b-c)}{c(a-b)} e^{-bt}$

$\omega_n Tr = 1.76\zeta^3 - 0.417\zeta^2 + 1.039\zeta + 1 -$

with Velocity feedback \rightarrow ss error finite w/ prop control
 0 w/ PI



$$C(s) = G(s)E(s)$$

$$E(s) = R(s) - C(s)$$

$$\frac{E(s)}{R(s)} = \frac{1}{1+G(s)}$$

with position feedback - s.s error with prop control = 0

$$r(t) = U_s(t) t^q$$

$q=0$ step
 $q=1$ ramp
 $q=2$ parabola

Type = # poles at origin.

Error	type 1	type 2	type 3
Step	$G \neq K_p$	$0 \neq K_p = \infty$	$0 \neq K_p = \infty$
Ramp	$\infty \neq K_v = 0$	$G \neq K_v$	$0 \neq K_v = \infty$
Parabola	$\infty \neq K_a = 0$	$\infty \neq K_a = 0$	$G \neq K_a$

$\omega = 2\pi f$	Step	$1/s$	G
	Ramp	$1/s^2$	t
	Parabola	$1/s^3$	$\frac{1}{2} t^2$

$$E(s) = \frac{R(s)}{1+G(s)}$$

$$e(\infty) = \lim_{s \rightarrow 0} \frac{sR(s)}{1+G(s)}$$

Static error const

pos-h

$$K_p = \lim_{s \rightarrow 0} G(s)$$

vel-h

$$K_v = \lim_{s \rightarrow 0} sG(s)$$

acc-h

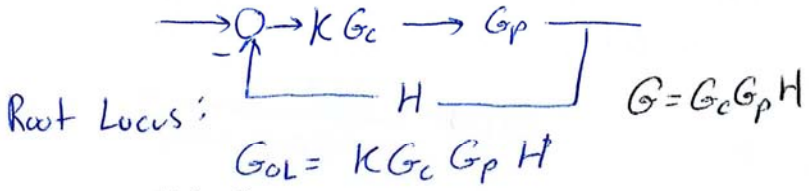
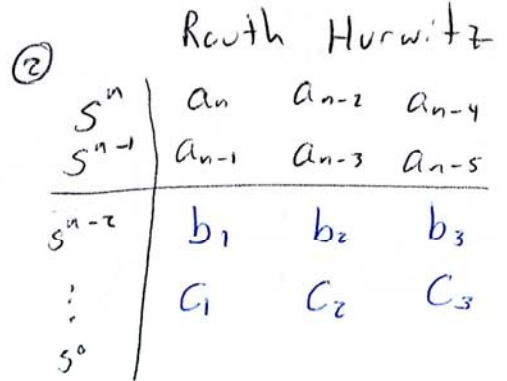
$$K_a = \lim_{s \rightarrow 0} s^2 G(s)$$

$$e_{step} = \frac{1}{1 + \lim_{s \rightarrow 0} G(s)} = \frac{1}{1 + K_p}$$

$$e_{ramp} = \frac{1}{\lim_{s \rightarrow 0} sG(s)} = \frac{1}{K_v}$$

$$e_{parabola} = \frac{1}{\lim_{s \rightarrow 0} s^2 G(s)} = \frac{1}{K_a}$$

① no sign changes.
 $a_3 s^3 + a_2 s^2 + a_1 s + a_0$
 $a_1 a_2 > a_3 a_0$



$$G_{OL} = KG_c G_p H$$

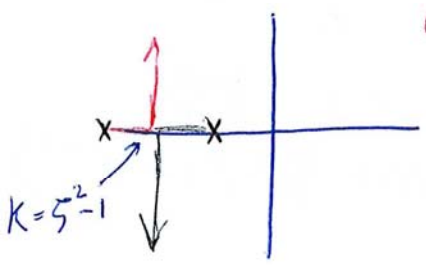
$$CL = \frac{KG_c G_p}{1 + KG_c G_p H}$$

CL char eqn
 $1 + KG(s) = 0$

$$b_1 = \frac{-1}{a_{n-1}} \left| \begin{array}{cc} a_n & a_{n-2} \\ a_{n-1} & a_{n-3} \end{array} \right|$$

$$b_2 = \frac{-1}{a_{n-1}} \left| \begin{array}{cc} a_n & a_{n-4} \\ a_{n-1} & a_{n-5} \end{array} \right|$$

$$c_1 = \frac{-1}{b_1} \left| \begin{array}{cc} a_{n-1} & a_{n-3} \\ b_1 & b_2 \end{array} \right|$$



① $|KG(s)| = 1$
 ② $\angle G(s) = -(2(n+1))\pi$
 $|G(s)| = 1/K$

③ Assume marginal stability

$$D(s) = (s+a)(s+j\omega)(s-j\omega)$$

$$= s^3 + as^2 + \omega^2 s + a\omega^2$$